

4.2 Conduction Heat Transfer

For one dimensional (1D) for homogeneous, isotropic solids:

Coordinate System	Governing Equation	Thermal condition	Examples
Cartesian	$\frac{\partial^2 T}{\partial x^2} + \frac{\dot{e}_{gen}}{k} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$	Transient	Slab, Plane wall
Cylindrical	$\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) + \frac{\dot{e}_{gen}}{k} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$	Transient	Solid/hollow pipe, tube, wire, etc.
Spherical	$\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial T}{\partial r} \right) + \frac{\dot{e}_{gen}}{k} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$	Transient	Solid & hollow sphere

\dot{e}_{gen} → Volumetric heat generation, W/m³
 k → Thermal conductivity, W/m.K
 α → Thermal diffusivity = $k/\rho c$, m²/s

4.2 Conduction Heat Transfer

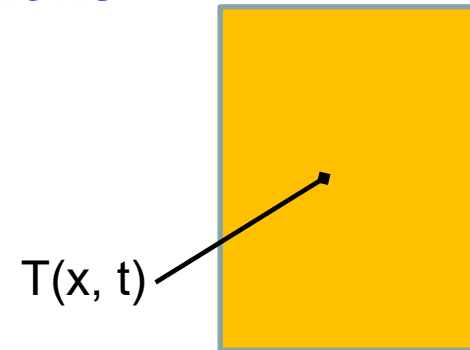
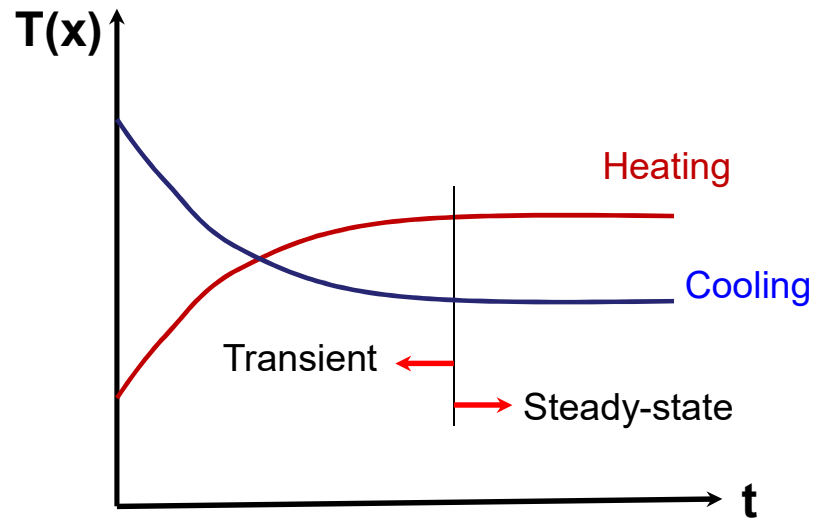
For one dimensional (1D) homogeneous, isotropic solids **without heat generation**:

Coordinate System	Governing Equation (Transient)	Governing Equation (Steady State)
Cartesian	$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$	$\frac{d^2 T}{dx^2} = 0$
Cylindrical	$\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) = \frac{1}{\alpha} \frac{\partial T}{\partial t}$	$\frac{d}{dr} \left(r \frac{dT}{dr} \right) = 0$
Spherical	$\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial T}{\partial r} \right) = \frac{1}{\alpha} \frac{\partial T}{\partial t}$	$\frac{d}{dr} \left(r^2 \frac{dT}{dr} \right) = 0$

$k \rightarrow$ Thermal conductivity, W/m.K
 $\alpha \rightarrow$ Thermal diffusivity = $k/\rho c$, m²/s

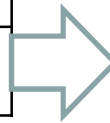
4.2 Conduction Heat Transfer

Transient versus Steady-State conditions



ME265: Thermal Engineering & Heat Transfer

Chapters
1. Energy Scenario
2. Thermodynamics
3. Mechanical Devices & Systems
4. Heat Transfer



4.1 Introduction	
4.2 Conduction	4.2.1 Conduction Equations 4.2.2 Boundary & Initial conditions 4.2.3 Steady Heat Conduction 4.2.4 Transient Heat Conduction
4.3 Convection	
4.4 Radiation	
4.5 Heat Exchanger	

4.2.2 Initial and Boundary Conditions

Conduction Equation/
Heat Diffusion Equation/
Governing Equation

Solution

Temperature distribution
within a solid

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$$

We need

$T(x,t)$

Two boundary conditions
One initial condition

$$\frac{d^2 T}{dx^2}$$

We need

$T(x)$

Two boundary conditions only

4.2.2 Initial and Boundary Conditions

2.2. Boundary and initial conditions

Initial condition:

Temperature in the medium at $t=0$: $T(\mathbf{x},0)$, $T(r,0)$

Boundary conditions:

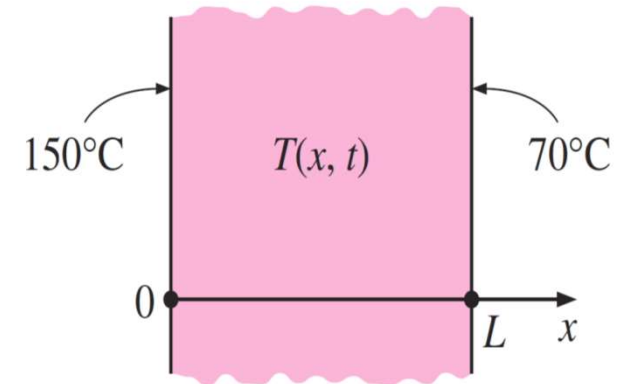
- Thermal conditions at the boundaries of the medium
- BCs are of various types
 - (1) Temperature BC—1st kind
 - (2) Heat flux BC—2nd kind
 - (3) Convection BC—3rd kind
 - (4) Interface BC

4.2.2 Initial and Boundary Conditions

(1) Temperature BC—1st kind

$$T(0, t) = T_1 = 150$$

$$T(L, t) = T_2 = 70$$



Mathematical formulation is given by:

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t} \quad \dots \dots \text{for } 0 < x < L, t > 0$$

$$T = 150 \quad \text{at } x=0$$

$$T = 70 \quad \text{at } x=L$$

$$T = F(x) \quad \text{at } t=0, 0 \leq x \leq L$$

4.2.2 Initial and Boundary Conditions

(2) Heat Flux BC—2nd kind

Values of q_0 and q_L are known.

$$-k \frac{\partial T(0, t)}{\partial x} = q_0$$

$$-k \frac{\partial T(L, t)}{\partial x} = q_L$$

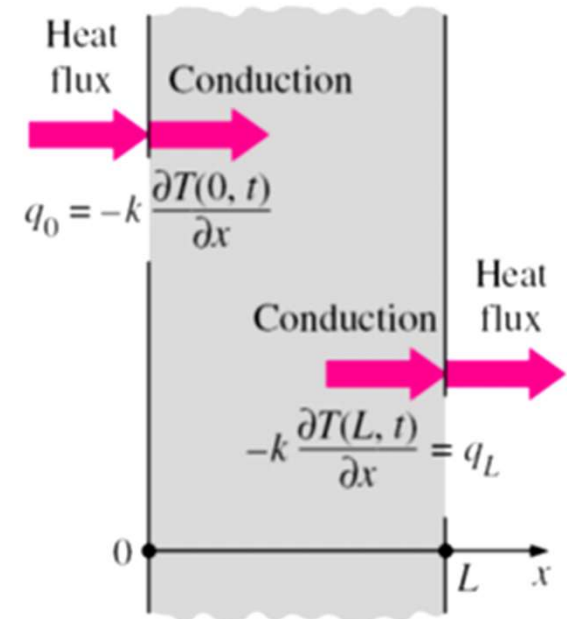
Mathematical formulation is given by:

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t} \quad \dots \dots \text{for } 0 < x < L, t > 0$$

$$-k \frac{\partial T}{\partial x} = q_0 \quad \text{at } x=0$$

$$-k \frac{\partial T}{\partial x} = q_L \quad \text{at } x=L$$

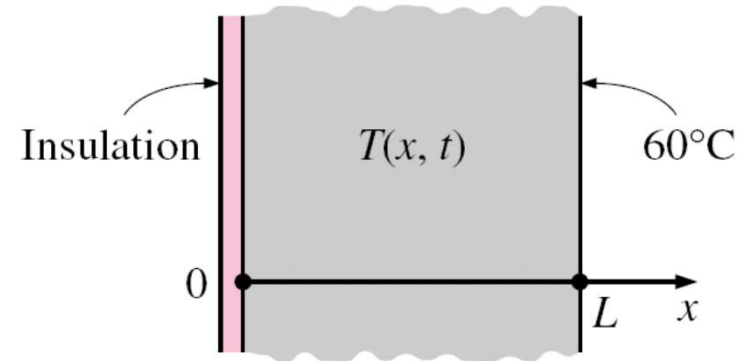
$$T = F(x) \quad \text{at } t=0, 0 \leq x \leq L$$



4.2.2 Initial and Boundary Conditions

(2) Heat Flux BC—2nd kind

Insulated boundary



Mathematical formulation is given by:

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t} \quad \dots \dots \text{for } 0 < x < L, t > 0$$

$$\frac{\partial T}{\partial x} = 0 \quad \text{at } x=0$$

$$T = 60 \quad \text{at } x=L$$

$$T = F(x) \quad \text{at } t = 0, 0 \leq x \leq L$$

4.2.2 Initial and Boundary Conditions

(2) Heat Flux BC—2nd kind

Thermal symmetry

Mathematical formulation is given by:

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t} \quad \dots \dots \text{for } 0 < x < L, t > 0$$

$$\frac{\partial T}{\partial x} = 0$$

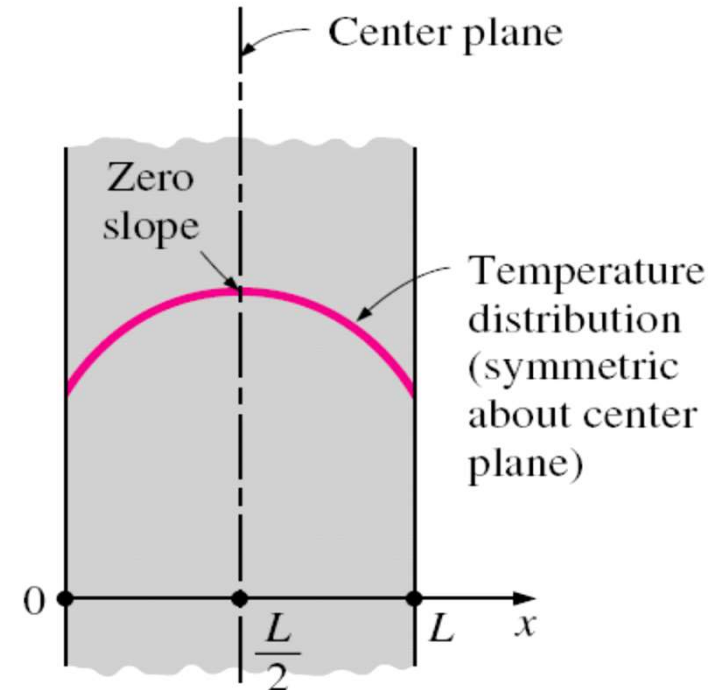
at $x=L/2$

.....?

at $x=L$

.....?

at $t = 0, 0 \leq x \leq L$



4.2.2 Initial and Boundary Conditions

(3) Convection BC—3rd kind

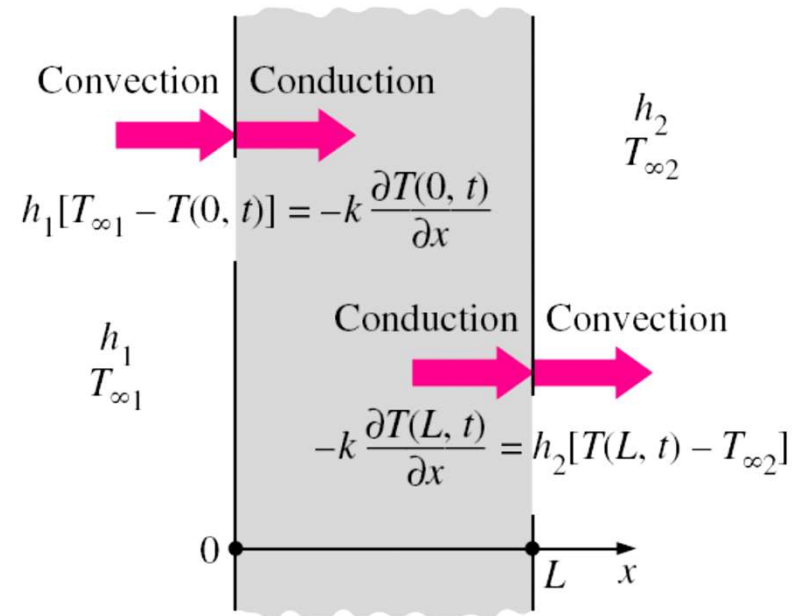
Mathematical formulation is given by:

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t} \quad \dots \dots \text{for } 0 < x < L, t > 0$$

$$-k \frac{\partial T}{\partial x} + h_1 T = h_1 T_{\infty 1} \quad \text{at } x=0$$

$$k \frac{\partial T}{\partial x} + h_2 T = h_2 T_{\infty 2} \quad \text{at } x=L$$

$$\dots \dots ? \quad \text{at } t=0, 0 \leq x \leq L$$



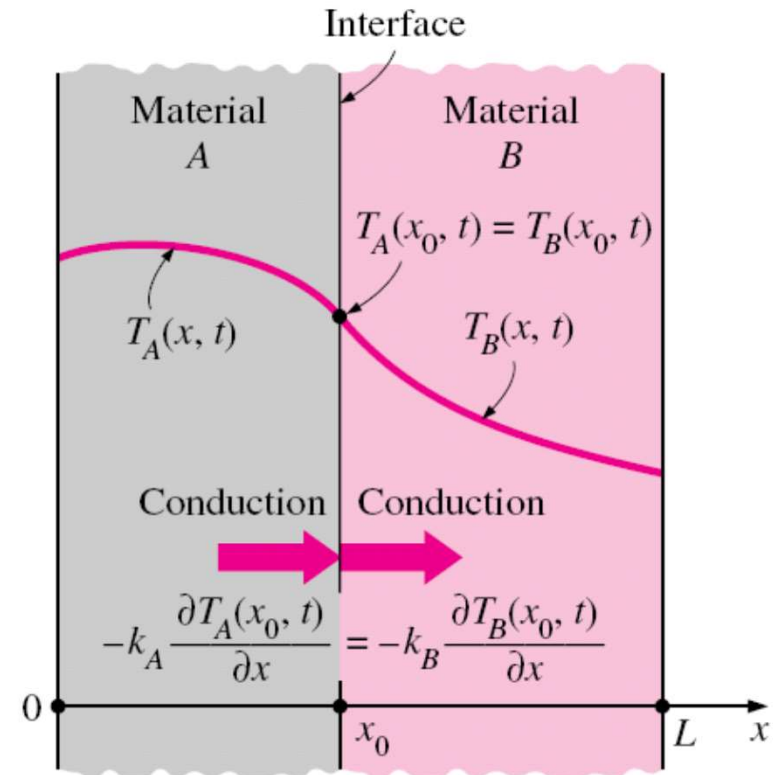
4.2.2 Initial and Boundary Conditions

(4) Interface BC

$$T_A(x_0, t) = T_B(x_0, t)$$

.....For perfect contact

$$-k_A \frac{\partial T_A(x_0, t)}{\partial x} = -k_B \frac{\partial T_B(x_0, t)}{\partial x}$$



If the perfect contact is not maintained,

- Contact conductance h_c is important
- $q = h_c(T_A - T_B)$ at x_0